# Internal pressures in buildings with a dominant opening and background porosity

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(Received December 18, 2011, Revised April 11, 2012, Accepted April 13, 2012)

Abstract. A dominant opening in a windward wall, which generates large internal pressures in a building, is a critical structural design criterion. The internal pressure fluctuations are a function of the dominant opening area size, internal volume size and external pressure at the opening. In addition, many buildings have background leakage, which can attenuate internal pressure fluctuations. This study examines internal pressure in buildings for a range of dominant opening areas, internal volume sizes and background porosities. The effects of background porosity are incorporated into the governing equation. The ratio of the background leakage area  $A_L$  to dominant opening area  $A_W$  is presented in a non-dimensional format through a parameter,  $\phi_6 - A_L / A_W$ . Background porosity was found to attenuate the internal pressure fluctuations when  $\phi_6$  is larger than 0.2. The dominant opening discharge coefficient, k was estimated to be between 0.05 to 0.40 and the effective background porosity discharge coefficient  $k'_L$ , was estimated to be between 0.05 to 0.50.

**Keywords:** background porosity; internal pressure; discharge coefficient; Helmholtz resonance; dominant opening; building

# 1. Introduction

The internal pressure in a building is dependent on the external pressure distribution, and the size and location of openings in the envelope. The pressure inside a nominally sealed building is generally smaller than the external pressure. However, the failure of a window or door can create a dominant windward wall opening and produce larger internal pressures; a critical design criterion.

The continuity of flow in and out of a building was used as the basis for specifying the mean internal pressure in a building by Liu (1975) and adopted in standards such as AS/NZS 1170.2 (2011). The fluctuating internal pressure in a building with a dominant opening was first studied by Holmes (1979) using the concept of the Helmholtz resonator. Further studies on model scale buildings with a dominant opening were carried out by Liu and Saathoff (1981), Liu and Rhee (1986), Stathopoulos and Luchien (1989) and Sharma and Richards (1997). The effects of sizes of dominant opening areas and volume on internal pressure were studied by Ginger et. al. (2008, 2010). Envelope flexibility has been considered by using an effective volume of the building as described by Vickery (1986). Ginger (2000) and Guha et. al. (2011a) have compared

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internal pressures measured in full scale buildings with analytical models. Guha *et al.* (2011b) incorporated flexibility and porosity explicitly in the analytical equations.

Vickery (1986, 1994) and Vickery and Bloxham (1992) studied the influence of background envelope porosity on internal pressure. They showed that porous openings exceeding nominally 10% of the dominant opening will attenuate the internal pressure fluctuations and porous openings larger than nominally 30% of the dominant opening will affect the mean internal pressure. Woods and Blackmore (1995) and Yu *et al.* (2008) also studied internal pressures in buildings with porosity and varying dominant opening areas and found that as the ratio of the leakage area to dominant opening area increases, the mean internal pressure is reduced. Oh (2007) developed a numerical model that treated each porous opening individually, using the power law equation, suggested by Shaw (1981). The power law assumes that the flow through porous openings is similar to flow in a pipe.

In the work described in this paper, internal pressure in a building with a range of volumes and windward wall openings with varying background porosities were studied using wind tunnel model tests and analytical methods. The variation of mean, fluctuations and peak internal pressures are presented in non-dimensional format that can be incorporated in building codes or design standards.

#### 2. Internal pressure

The flow through an opening in a building can be described using the discharge equation shown in Eq. (1). Here,  $p_e$  is the external pressure at the opening and  $p_i$  is the internal pressure in the building,  $\rho$  is the density of air, U and  $\dot{U}$  are the area averaged flow velocity and acceleration through the opening, respectively. The first term on the right side of the equation characterizes the losses and the second term is the inertial term describing a "slug" of air passing through the opening. The loss coefficient  $C_L$  can be represented as  $1/k^2$ , where k is the discharge coefficient and  $l_e$  is the effective length of the slug of air.

$$p_e - p_i = \frac{1}{2}\rho C_L U^2 + \rho l_e \dot{U} \tag{1}$$

Vickery and Bloxham (1992), noted that  $C_L$  and  $l_e$  can only be defined for situations such as steady flow through sharp edged circular openings connecting large volumes, where potential flow theory gives  $C_L = 2.68$  (k=0.6) and  $l_e = C_I \sqrt{A}$ . Here, A is the opening area, and the inertial coefficient  $C_I$ , is  $\sqrt{\pi/4} = 0.89$ . However, since potential flow conditions do not apply to highly turbulent flow through a building opening, experimental data must be used to estimate values for k and  $C_I$ . Previous studies have suggested a range of values for k from 0.15 to 1.0 and  $C_I$  of up to 2.

The pressure on the building p(t), varying with time t, is expressed as a pressure coefficient,

 $C_p(t) = p(t) / \frac{1}{2} \rho \overline{U}_h^2$ , where  $\overline{U}_h$  is the mean wind speed at a height *h*. The statistical

characteristics of the time varying pressure are presented as the mean  $\overline{C}_p$ , standard deviation

 $C_{\sigma p}$  and maximum  $\hat{C}_{p}$ .

The principle of conservation of mass is combined with Eq. (1) to give the mean internal pressure in a building with windward and leeward opening areas,  $A_W$  and  $A_L$  respectively, as shown in Eq. (2).

$$\overline{C}_{pi} = \frac{C_{pW}}{I + \left(\frac{A_L}{A_W}\right)^2} + \frac{C_{pL}}{I + \left(\frac{A_W}{A_L}\right)^2}$$
(2)

Here,  $\overline{C}_{pi}$  is the mean internal pressure in the building and  $\overline{C}_{pW}$  and  $\overline{C}_{pL}$  are the mean external pressure coefficients at the windward and leeward openings respectively.

Holmes (1979) and Vickery (1986) showed that the internal pressure fluctuations in a building with a dominant windward opening area  $A_W$ , are generated by a "slug" of air moving in and out of the opening and can be described by Eq. (3). Here,  $V_e$  is the effective internal volume, and  $a_s$  is

the speed of sound. The undamped Helmholtz frequency is  $f_H = \frac{1}{2\pi} \sqrt{a_s^2 A_W / l_e V_e}$ .

$$\frac{l_e V_e}{a_s^2 A_W} \ddot{C}_{pi} + \left(\frac{V_e \overline{U}_h}{2a_s^2 k A_W}\right)^2 \dot{C}_{pi} \left| \dot{C}_{pi} \right| + C_{pi} = C_{pW}$$
(3)

Using dimensional analysis, Holmes (1979) represented Eq. (3) in terms of five nondimensional parameters

$$\phi_1 = \frac{A_W^{3/2}}{V_e} \qquad \phi_2 = \frac{a_s}{\overline{U}_h} \qquad \phi_3 = \frac{\rho \overline{U}_h \sqrt{A_W}}{\mu} \qquad \phi_4 = \frac{\sigma_u}{\overline{U}} \qquad \phi_5 = \frac{\lambda_u}{\sqrt{A_W}}$$

where  $\mu$  is viscosity of air and  $\lambda_u$  is the integral length scale of turbulence. By including a dimensionless time parameter  $t^* = t\overline{U}_h / \lambda_u$ , Eq. (3) can be expressed in non-dimensional format, Eq. (4).

$$\frac{C_I}{\phi_1 \phi_2^2 \phi_5^2} \frac{d^2 C_{pi}}{dt^{*2}} + \frac{1}{4k^2} \left(\frac{1}{\phi_1 \phi_2^2 \phi_5}\right)^2 \frac{dC_{pi}}{dt^*} \left|\frac{dC_{pi}}{dt^*}\right| + C_{pi} = C_{pW}$$
(4)

The  $\phi_1 \phi_2^2$  parameters in the first and second terms of Eq. (4) can be replaced by the nondimensional parameter  $S^* = (A_W^{3/2}/V_e) \times (a_s/\overline{U}_h)^2$  as shown by Ginger *et al.* (2008, 2010). Eq. (5) shows the governing equation in non-dimensional  $S^*$  format. This implies that internal pressure fluctuations depend on the size of the opening area with respect to the size of the internal volume, and that there is a unique solution for  $C_{pi}$  for a given  $S^*$  and  $\phi_S$ , if k and  $C_I$  are known.

$$\frac{C_{I}}{S^{*}\phi_{5}^{2}}\frac{d^{2}C_{pi}}{dt^{*2}} + \left(\frac{1}{4k^{2}}\right)\left(\frac{1}{S^{*}\phi_{5}}\right)^{2}\frac{dC_{pi}}{dt^{*}}\left|\frac{dC_{pi}}{dt^{*}}\right| + C_{pi} = C_{pW}$$
(5)

#### 2.1 Background porosity

The previous section described the internal pressure response of a sealed building (no background porosity) with a dominant opening, however buildings will have varying degrees of background porosity. The background porosity in a building is generally difficult to quantify, and it is reasonable to assume a single effective lumped leakage area (Vickery and Bloxham 1992). Vickery (1986) showed that the inertial term for porous openings is orders of magnitude smaller than the damping term and can be ignored. Yu *et al.* (2008) and Guha *et al.* (2009) produced detailed derivations incorporating a lumped porous leeward opening area in the governing equation.

Ignoring the inertial term in Eq. (1), the air flow through an effective porous opening can be shown as  $p_i - \overline{p}_L = \frac{1}{2}\rho C'_L U^2$ , where  $\overline{p}_L$  is the mean pressure acting on the leeward (i.e., porous) surfaces, and  $C'_L$  is the effective loss coefficient for the lumped porous opening and can be related to an effective discharge coefficient  $k'_L$ , by  $C'_L = (1/k'_L)^2$ . The parameter, *n*, is equal to the ratio of specific heats (1.4) for an adiabatic process. Using the continuity equation, the velocity, *U*, and the acceleration,  $\dot{U}$  through the dominant opening can be found by Eqs. (6) and (7) respectively.

$$U = \frac{1}{2} \frac{\rho \overline{U}_h^2}{n p_i} \frac{V_e}{A_W} \dot{C}_{pi} + \frac{A_L}{A_W} U_L$$
(6)

$$\dot{U} = \frac{1}{2} \frac{\rho \overline{U}_h^2}{n p_i} \frac{V_e}{A_W} \ddot{C}_{pi} + \frac{A_L}{A_W} \dot{U}_L \tag{7}$$

 $U_L$  and  $\dot{U}_L$  are the velocity and acceleration through the lumped leeward opening respectively, which are derived from the Bernoulli Equation.  $U_L$  and  $\dot{U}_L$  are represented by

$$U_{L} = \overline{U}_{h} \sqrt{\frac{(C_{pi} - \overline{C}_{\rho L})}{C_{L}'}} \qquad \dot{U}_{L} = \frac{\overline{U}_{h} \dot{C}_{pi}}{\left(2 \times \sqrt{(C_{pi} - \overline{C}_{pL})C_{L}'}\right)}$$

The internal pressure fluctuations in a building, with a dominant windward opening and lumped leeward opening, are obtained by combining Eqs. (6), (7) and (3), to give Eq. (8). It should be noted that a sealed building case (when  $A_L=0$ ), gives Eq. (3).

$$\frac{l_e V_e}{a_s^2 A_W} \ddot{C}_{pi} + \frac{k_L' l_e A_L}{\overline{U}_h A_W} \cdot \frac{\dot{C}_{pi}}{\sqrt{(C_{pi} - \overline{C}_{pL})}} + \frac{\rho^2 \overline{U}_h^2 V_e^2}{4k^2 n^2 p_i^2 A_W^2} \cdot \left(\frac{2k_L' A_L n p_i}{V_e \rho \overline{U}_h} \cdot \sqrt{(C_{pi} - \overline{C}_{pL})} + \dot{C}_{pi}\right) (8)$$

$$\left|\frac{2k'_{L}A_{L}np_{i}}{V_{e}\rho\overline{U}_{h}}\cdot\sqrt{(C_{pi}-\overline{C}_{pL})}+\dot{C}_{pi}\right|+C_{pi}=C_{pW}$$

The magnitude of background porosity in the building can be expressed by a non-dimensional parameter  $\phi_6 = A_L / A_W$ . Eq. (8) can then be represented in non-dimensional format shown in Eq. (9).

$$\frac{C_{I}}{S^{*}\phi_{5}^{2}}\frac{d^{2}C_{pi}}{dt^{*2}} + k_{L}^{\prime}\frac{\phi_{6}}{\phi_{5}}\frac{C_{I}}{\sqrt{(C_{pi}-\overline{C}_{pL})}}\frac{dC_{pi}}{dt^{*}} + \frac{k_{L}^{\prime}}{k^{2}}\phi_{6}^{2}(C_{pi}-\overline{C}_{pL}) + \frac{k_{L}^{\prime}}{k^{2}}\frac{\phi_{6}}{S^{*}\phi_{s}}\sqrt{C_{pi}-\overline{C}_{pL}}\frac{dC_{pi}}{dt^{*}} + \left(\frac{1}{4k^{2}}\right)\left(\frac{1}{S^{*}\phi_{s}}\right)^{2}\frac{dC_{pi}}{dt^{*}}\left|\frac{dC_{pi}}{dt^{*}}\right| + C_{pi} = C_{pW}$$
(9)

# 3. Experimental and numerical methods

A series of model scale tests were carried out on a 400 mm  $\times$  200 mm  $\times$  100 mm building, in the wind tunnel at the School of Engineering and Physical Sciences at James Cook University. The model was extended 600 mm below the wind tunnel floor to allow the varying of internal volume by 3, 5 and 7 times the normal volume of the building shown in Fig. 1(a). Fig. 1(b) shows the layout of the building porosity. Background porosities *P1*, *P2*, *P3* and *P4* were modelled with 64  $\times$ 1.5 mm diameter holes and 54  $\times$  3 mm diameter holes installed uniformly on the two side walls, the leeward wall and the roof, which could be blocked or opened as needed.

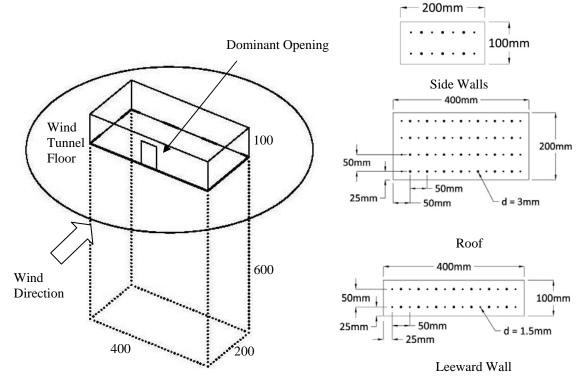
The windward external pressure coefficients  $C_{pW}$ , were measured on 30 taps across a door centred on the 400mm windward wall. The pressures were area averaged for four dominant opening areas: A1 (20mm × 20mm), A2 (25mm × 50mm), A3 (50mm × 50mm) and A4 (80mm × 50mm), shown in Fig. 1(c). The leeward pressure  $C_{pL}$  was obtained by area averaging pressures measured on the roof, side walls and leeward wall.

The internal pressure  $C_{pi}$  was measured in the model for each combination of windward wall opening areas: A1, A2, A3, A4, volumes: V3, V5, V7 and background porosities: P1, P2, P3, P4 given in Table 1. Each windward opening area, porosity and volume gives a set of  $S^*$ ,  $\phi_5$  and  $\phi_6$  values.

Generally all of the non-dimensional parameters cannot be matched between full scale and model scale as needed for similarity requirements. The effects of Reynolds number mismatch are not explicitly accounted for in this study. This is assumed to have a negligible effect for the dominant opening sizes considered here, but may influence the discharge coefficients for the background porosity. The results herein provide a general trend of the effects of background porosity.

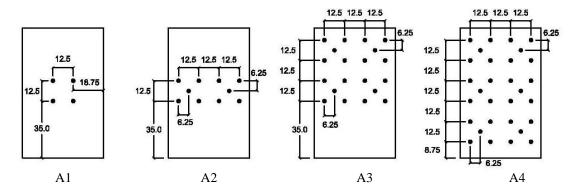
The atmospheric boundary layer was scaled at 1/200 for Terrain Category 2 as defined by AS/NZS 1170.2 (2011). Tests were conducted at a mean approach wind speed of approximately 10 m/s at roof height,  $\overline{U}_h$  (height = 100 mm model scale). The integral length scale of turbulence  $\lambda_u$  at roof height was estimated to be 300 mm. Pressures were sampled at 1250 Hz for 30 seconds using dynamic pressure measurement system from Turbulent Flow Instrumentation. The experimental setup is similar to that described by Ginger *et al.* (2010).

The numerical model described by Ginger *et al.* (2008, 2010) was used to simulate internal pressure time histories for the range of volume and dominant opening sizes with varying porosity. The measured external pressures at the dominant opening were used to solve Eq. (8) by a first order explicit finite difference scheme



(a) Wind tunnel model with 600mm extension below the floor

(b) Simulated porosity on leeward surfaces



(c) Varying opening area and external pressure taps used to windward wall pressures (dimensions in mm)

Fig. 1 200 mm x 400 mm x 100 mm wind tunnel model

| $A_W$ (mm <sup>2</sup> )               | Volume (mm <sup>3</sup> ) | $S^{*}$ | $A_L (\mathrm{mm}^2)$ | $\phi_6$ |
|--|---------------------------|---------|-----------------------|----------|
| $A1 = 20 \times 20$<br>$\phi_5 = 15.0$ |                           | 0.46    | P1 = 0                | 0        |
|  | V/2 200 100 200           |         | P2 = 113              | 0.28     |
|  | <i>V3</i> = 200×400×300   |         | P3 = 382              | 0.95     |
|  |                           |         | P4 = 495              | 1.24     |
|  | <i>V5</i> = 200×400×500   | 0.25    | P1 = 0                | 0        |
|  |                           |         | P2 = 113              | 0.28     |
|  |                           |         | P3 = 382              | 0.95     |
|  |                           |         | P4 = 495              | 1.24     |
|  | <i>V7</i> = 200×400×700   | 0.17    | P1 = 0                | 0        |
|  |                           |         | P2 = 113              | 0.28     |
|  |                           |         | P3 = 382              | 0.95     |
|  |                           |         | P4 = 495              | 1.24     |
| $A2 = 50 \times 25$<br>$\phi_5 = 8.5$  | <i>V3</i> = 200×400×300   | 2.56    | P1 = 0                | 0        |
|  |                           |         | P2 = 113              | 0.09     |
|  |                           |         | P3 = 382              | 0.31     |
|  |                           |         | P4 = 495              | 0.40     |
|  | V5 = 200×400×500          | 1.38    | P1 = 0                | 0        |
|  |                           |         | P2 = 113              | 0.09     |
|  |                           |         | P3 = 382              | 0.31     |
|  |                           |         | P4 = 495              | 0.40     |
|  | V7 = 200×400×700          | 0.95    | P1 = 0                | 0        |
|  |                           |         | P2 = 113              | 0.09     |
|  |                           |         | P3 = 382              | 0.31     |
|  |                           |         | P4 = 495              | 0.40     |
| $A3 = 50 \times 50$ $\phi_5 = 6.0$     | <i>V3</i> = 200×400×300   | 7.24    | P1 = 0                | 0        |
|  |                           |         | P2 = 113              | 0.05     |
|  |                           |         | P3 = 382              | 0.15     |
|  |                           |         | P4 = 495              | 0.20     |
|  | V5 = 200×400×500          | 3.90    | P1 = 0                | 0        |
|  |                           |         | P2 = 113              | 0.05     |
|  |                           |         | P3 = 382              | 0.15     |
|  |                           |         | P4 = 495              | 0.20     |
|  | V7 = 200×400×700          | 2.69    | P1 = 0                | 0        |
|  |                           |         | P2 = 113              | 0.05     |
|  |                           |         | P3 = 382              | 0.15     |
|  |                           |         | P4 = 495              | 0.20     |
| $A4 = 80 \times 50$<br>$\phi_5 = 4.7$  | <i>V3</i> = 200×400×300   | 14.65   | P1 = 0                | 0        |
|  |                           |         | P2 = 113              | 0.03     |
|  |                           |         | P3 = 382              | 0.09     |
|  |                           |         | P4 = 495              | 0.12     |
|  | <i>V5</i> = 200×400×500   | 7.90    | P1 = 0                | 0        |
|  |                           |         | P2 = 113              | 0.03     |
|  |                           |         | P3 = 382              | 0.09     |
|  |                           |         | P4 = 495              | 0.12     |
|  | V7 = 200×400×700          | 5.44    | P1 = 0                | 0        |
|  |                           |         | P2 = 113              | 0.03     |
|  |                           |         | P3 = 382              | 0.09     |
|  |                           |         | P4 = 495              | 0.12     |

Table 1 Test configurations opening size, volume size,  $S^*$ , leakage area  $A_L$  with corresponding  $\phi_5$ , and  $\phi_6$ 

### 4. Results and discussions

The mean, standard deviation and maximum internal and external pressures were measured for each test configuration described in Table 1. The mean external pressure on the windward and leeward surfaces was used to predict the mean internal pressure using Eq. (2). Fig. 2 compares the measured and theoretical  $\overline{C}_{pi}$  to  $\overline{C}_{pW}$  ratios from Eq.(2), with varying effective leeward openings,  $\phi_6$ . The theoretical results show that when  $\phi_6$  is less than approximately 0.2,  $\overline{C}_{pi}$  is within 10% of  $\overline{C}_{pW}$ . However, when  $\phi_6$  is larger than 0.2, the mean internal pressures are significantly reduced in comparison with the external pressures at the windward opening. The measured mean pressures follow the same trends and are similar to those obtained by Vickery (1994).

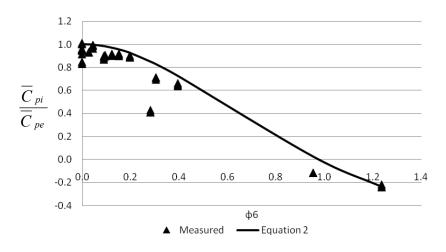


Fig. 2 Ratio of internal and external mean pressure coefficients to  $\phi_6$ 

The ratio of the measured  $C_{\sigma pi}$  to  $C_{\sigma pW}$  versus  $S^*$  are presented for ranges of  $\phi_6$  in Fig. 3. When  $\phi_6$  is less than 0.1, the internal pressure fluctuations were greater than the external pressure fluctuations for  $S^*$  values larger than 0.5. For cases when  $S^*$  is less than 0.5, the internal pressure fluctuations are greatly reduced compared to external pressure fluctuations. This figure also shows that when the background porosity is increased the internal pressure fluctuations are attenuated. The magnitude of the reduction is dependent on the magnitude of the porous area to dominant opening area,  $\phi_6$ . For low  $\phi_6$  values between 0.1 and 0.2, the attenuation of the internal pressure fluctuations decrease by greater than 20%, especially when  $S^*$  is less than 0.5. The ratio of the measured  $\hat{C}_{pi}$  to  $\hat{C}_{pW}$  is shown in Fig. 4. The measured peak pressures follow similar trends to the measured standard deviations. The results for the sealed building case are similar to those found by Ginger *et* 

al. (2010).

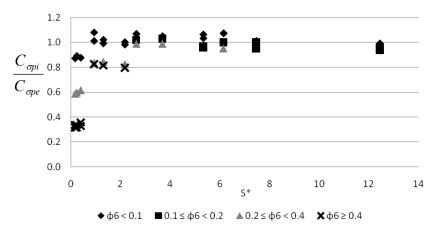


Fig. 3 Ratio of internal and external pressure standard deviation versus S\* for varying  $\phi_6$ 

The measured external and internal pressure spectra,  $S_{Cp}(f)$ , with varying background porosities P1, P2, P3 and P4, for opening area A2 with volumes V3 and V7, are shown in Figs. 5 (a) and (b), and opening area A4 with volumes V3 and V7, are shown Fig. 6 (a) and (b). When the building is sealed (P1), Figs. 5 (a) and (b) show a distinct Helmholtz resonance at approximately 45 Hz and 40 Hz respectively. In both cases, increasing the porosity from P2, P3 to P4 progressively reduces the magnitude of the Helmholtz peak. This increased damping is caused by large values of  $\phi_6$  in the second and third terms of Eq. (9). For this case  $\phi_6$  was equal to 0.09, 0.31 and 0.40 for porosities P2, P3 and P4 respectively. When  $\phi_6$  is equal to 0.31 and 0.40, the Helmholtz resonance is almost completely damped. Similar trends are observed in Figs. 6 (a) and (b), however there is less attenuation of the Helmholtz peaks due to smaller  $\phi_6$  values. The  $\phi_6$ values in Figs. 6(a) and (b) were 0.03, 0.09 and 0.12 for P2, P3 and P4 respectively.

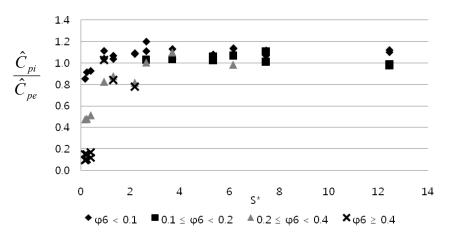


Fig. 4 Ratio of internal and external peak pressure coefficient versus S\* for varying  $\phi_6$ 

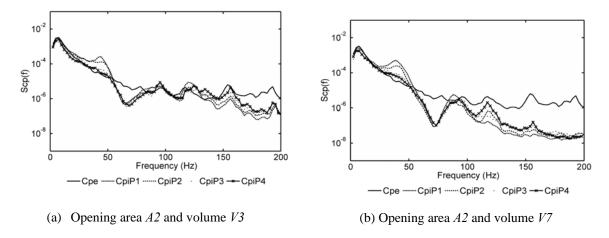


Fig. 5 Measured external and internal pressure spectra for varying porosities *P1*, *P2*, *P3* and *P4* for area *A2* and volumes *V3* and *V7* 

Simulated internal pressure spectra generated from the numerical model were matched with measured internal pressure spectra obtained from the wind tunnel results by varying the parameters:  $C_I$ , k and  $k'_L$ . The inertial coefficient  $C_I$  was estimated to be about 2.0 by matching the Helmholtz frequencies. Vickery (1994) also stated that  $C_I$  could be up to a value of 2 for highly fluctuating flow. Guha *et al.* (2009) and Sharma and Richards (1997) used a separate flow coefficient to account for the contraction of the air slug at the opening that gives an overall equivalent  $C_I$  of approximately 1.5. The windward wall opening discharge coefficient k, was estimated using the sealed building case (*P1*) and matching the magnitude of the Helmholtz resonance peak. k was assumed to remain constant for all porous cases. The background leakage discharge coefficient  $k'_L$  was adjusted to match internal pressure spectra for *P2*, *P3* and *P4*. The measured and simulated internal pressure spectra for configurations A2-V7 and A4-V7 with porosities of *P1*, *P2*, *P3* and *P4* are presented in Figs. 7 and 8, respectively. The simulations show that the numerical model matches the measured wind tunnel results.

The estimated values of k range from 0.1 to 0.4 for various  $S^*$ , shown in Fig. 9. The discharge coefficient for the dominant opening, k, decreases when  $S^*$  increases. This suggests that when the opening area is small and the volume is large, the airflows through the orifice more freely, similar to flow through an orifice connecting two infinitely large volumes. Conversely when the opening area is large and the volume is small, the flow is impeded and results in a smaller discharge coefficient. Similar findings were published by Ginger *et al.* (2010). Recent experiments by Kim and Ginger (2012) showed that when air flow oscillates in and out of an orifice, the values of k can be reduced to within this range.

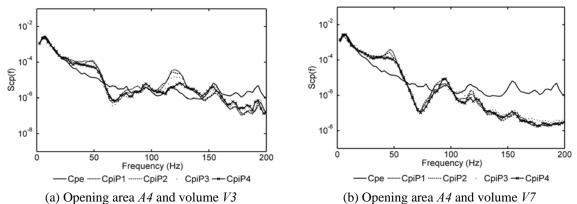


Fig. 6 Measured external and internal pressure spectra for varying porosities P1, P2, P3 and P4 for area A4 and volumes V3 and V7

Fig. 10 shows the effective discharge coefficient  $k'_L$  for the background leakage area with increasing  $S^*$  and varying  $\phi_6$ . The estimated values of  $k'_L$  have a large variation, ranging from 0.05 to 0.5. Fig. 10 shows that in general, for larger  $\phi_6$  values, the estimated  $k'_L$  values tend to decrease. This may be explained by the fact that the increase in porosity results in a reduction in internal pressure and a lower pressure drop across the leeward surfaces. This results in a small discharge coefficient.

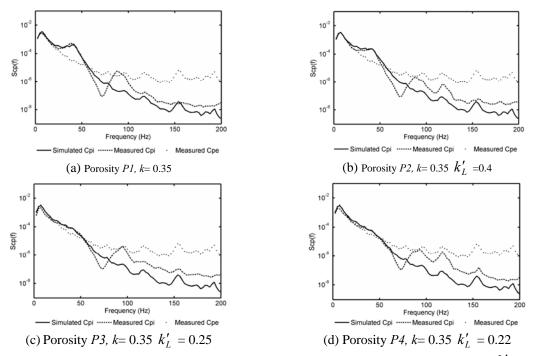


Fig. 7 Area A2 and internal volume V7 for porosities with opening k = 0.35 and varying  $k'_L$ 

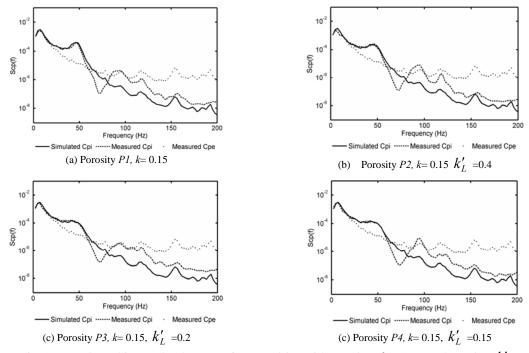


Fig. 8 Area A4 and internal volume V7 for porosities with opening k = 0.15 and varying  $k'_L$ 

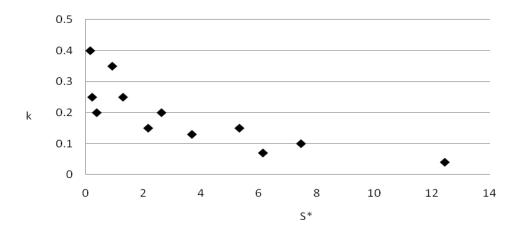


Fig. 9 Discharge coefficient for dominant opening k with increasing  $S^*$  for  $\phi_6 = 0$ 

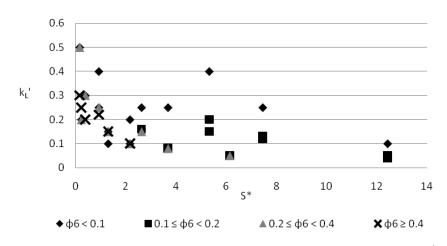


Fig. 10 Discharge coefficient for background porosity  $k'_L$  with increasing  $S^*$ 

# 5. Conclusions

The effects of internal pressure fluctuations in buildings with a range of internal volumes, dominant opening areas and background porosities were studied using a combination of analytical methods and a series of wind tunnel model experiments. The mean, standard deviation and maximum internal pressures were compared to the corresponding external pressure at the windward dominant opening and presented using a series of non-dimensional parameters: the dominant opening area to volume parameter  $S^* = (a_s / \overline{U}_h)^2 (A_W^{3/2} / V_e)$  and background porous area to dominant opening area parameter  $\phi_6 = A_L / A_W$ . The discharge coefficients for the dominant opening k and effective discharge coefficient for the lumped background porosity area  $k'_L$  were estimated by matching spectra. The study found that :

• The non-dimensional form of the governing equation for internal pressure can be modified to incorporate an effective lumped background leakage area by a dimensionless  $\phi_6 = A_L / A_W$  parameter.

• For background porosity cases where  $\phi_6$  is less than 0.2, has is minimal affect on the  $\overline{C}_{pi}$ .

However, when  $\phi_6$  is greater than 0.2, a reduction in  $\overline{C}_{pi}$  greater than 10% can be expected.

• Internal pressure fluctuations are influenced by dominant opening size, internal volume size and amount of background porosity.

•  $C_{\sigma p i}$  are attenuated by larger than 20% when  $\phi_6$  is larger than 0.2.

• The dominant opening discharge coefficient k, is dependent on the sizes of the opening area and internal volume.

• Discharge coefficients were estimated to be between 0.05 to 0.40. These values are lower than the theoretical value of 0.61 for potential flow. Similar trends were found by Ginger *et al.* (2010). The effective discharge coefficient of the lumped porous area  $k'_L$  was estimated to be in the range of 0.05 and 0.50.

• The inertial coefficient  $C_I$ , of 2 satisfactorily predicts the Helmholtz frequency.

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